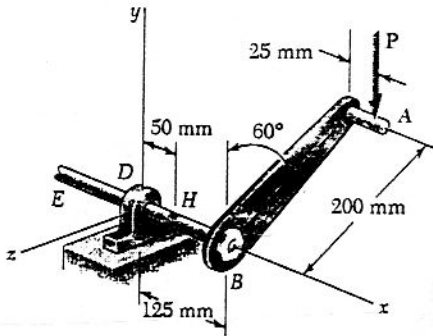


Problem 8.41

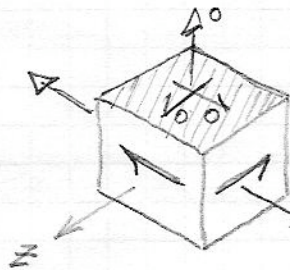
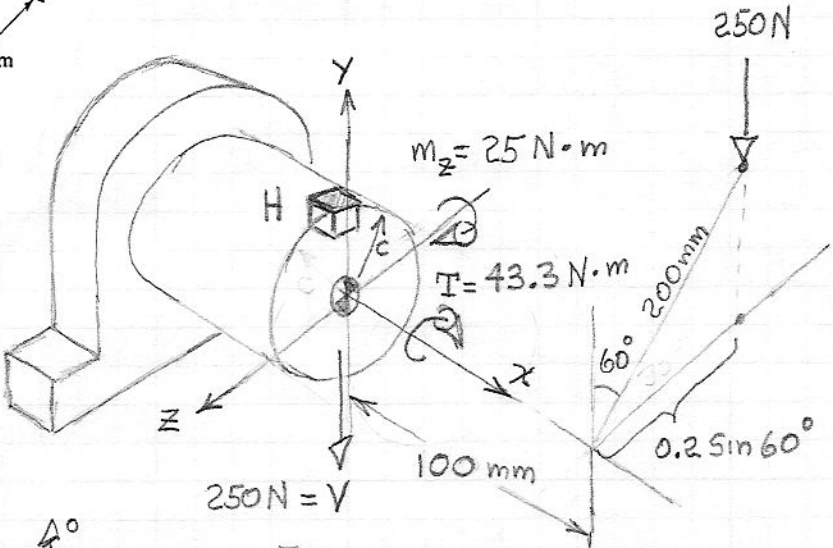
8.41 A vertical force P of magnitude 250 N is applied to the crank at point A . Knowing that the shaft BDE has a diameter of 18 mm, determine the principal stresses and the maximum shearing stress at point H located at the top of the shaft, 50 mm to the right of support D .



$$C = \frac{0.018}{2} = 0.009 \text{ m}$$

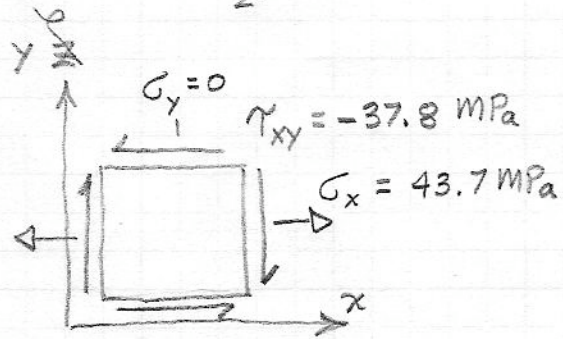
$$I_x = \frac{\pi}{4} C^4 = 5.153 \times 10^{-9} \text{ m}^4$$

$$J = 2I = 10.31 \times 10^{-9} \text{ m}^4$$



$$\tau_{xy} = -\frac{Tc}{J} = -\frac{43.3(0.09)}{10.31 \times 10^{-9}} \quad \tau_{xy} = -37.8 \text{ MPa}$$

$$\sigma_x = \frac{M_z c}{I_x} = \frac{25(0.09)}{5.153 \times 10^{-9}} \quad \sigma_x = 43.7 \text{ MPa}$$



$$\sigma_{avg} = \frac{\sigma_x + \sigma_y}{2} = 21.8 \text{ MPa}$$

$$R = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$R = \sqrt{\left(\frac{43.7}{2}\right)^2 + (37.8)^2} = 43.7 \text{ MPa}$$

$$\sigma_{max} = \sigma_{avg} + R \quad \sigma_{min} = \sigma_{avg} - R$$

$$\sigma_{max} = 65.5 \text{ MPa}$$

$$\sigma_{min} = -21.8 \text{ MPa}$$

$$\tau_{max} \text{ (in-plane)} = R$$

$$\tau_{max} = 43.7 \text{ MPa}$$